

Amendments to the Claims:

[1] (Currently Amended) A method of controlling pressure in an electric injection molding machine, comprising:

detecting an angular velocity  $\omega$  of a motor operative to propel forward a screw in an injection molding machine;

deriving an estimated melt pressure value  $\hat{\delta}$  without deriving a differential of the detected angular velocity  $\omega$ , based on an observer, from said detected angular velocity  $\omega$  of said motor and a torque command value  $T^{CMD}$  given to said motor; and

controlling said motor such that said estimated melt pressure value  $\hat{\delta}$  follows a melt pressure setting  $\delta^{REF}$ .

[2] (Original) The method of controlling pressure in an electric injection molding machine according to claim 1, wherein said observer is represented by the following Expression 1.

[Expression 1]

$$\frac{d}{dt} \begin{pmatrix} \omega^{\wedge} \\ \delta^{\wedge} \end{pmatrix} = \begin{pmatrix} d_1 & 1/J \\ d_2 & 0 \end{pmatrix} \begin{pmatrix} \omega^{\wedge} \\ \delta^{\wedge} \end{pmatrix} + \begin{pmatrix} 1/J \\ 0 \end{pmatrix} T^{CMD} + \begin{pmatrix} 1/J \\ 0 \end{pmatrix} F(\omega) - \begin{pmatrix} d_1 \\ d_2 \end{pmatrix} \omega$$

where  $\omega^{\wedge}$ : Estimated value of Angular velocity of Motor

$d_1, d_2$ : Certain coefficients

J: Inertia moment over Injection mechanism

F( $\omega$ ): Dynamic frictional resistance and Static frictional resistance over Injection mechanism

[3] (Original) The method of controlling pressure in an electric injection molding machine according to claim 1, wherein said observer is represented by the following Expression 2.

$$\begin{aligned}\omega^{\wedge} &= \omega^{\wedge}_{-1} + \{d_1(\omega^{\wedge}_{-1} - \omega) + (1/J)(T^{CMD}_{-1} + \delta^{\wedge}_{-1} + F(\omega))\} dt \\ \delta^{\wedge} &= \delta^{\wedge}_{-1} + \{d_2(\omega^{\wedge}_{-1} - \omega)\} dt\end{aligned}$$

[Expression 2]

where  $\omega^{\wedge}$ : Estimated value of Angular velocity of Motor

$d_1, d_2$ : Certain coefficients

J: Inertia moment over Injection mechanism

F( $\omega$ ): Dynamic frictional resistance and Static frictional resistance over Injection mechanism

$x_{-1}$ : Value of x at Immediately preceding processing period

[4] (Original) The method of controlling pressure in an electric injection molding machine according to claim 1,

wherein said screw in said injection molding machine and said motor are coupled together via a belt suspended around pulleys mounted on respective rotation shafts, and

wherein said observer is represented by the following Expression 3.

[Expression 3]

$$\frac{d}{dt} \begin{pmatrix} \hat{\omega}^M \\ \hat{\omega}^L \\ \hat{F} \\ \hat{\sigma} \\ \hat{\sigma} \end{pmatrix} = \begin{pmatrix} d_1 & 0 & -\frac{R^M}{J^M} & 0 & 0 \\ d_2 & 0 & \frac{R^L}{J^L} & \frac{1}{J^L} & 0 \\ d_3 + K_b R^M & -K_b R^L & 0 & 0 & 0 \\ d_4 & K_w & \frac{K_{wd} R^L}{J^L} & \frac{K_{wd}}{J^L} & 1 \\ d_5 & 0 & 0 & 0 & 0 \end{pmatrix} \begin{pmatrix} \hat{\omega}^M \\ \hat{\omega}^L \\ \hat{F} \\ \hat{\sigma} \\ \hat{\sigma} \end{pmatrix} + \begin{pmatrix} \frac{1}{J^M} \\ 0 \\ 0 \\ 0 \\ 0 \end{pmatrix} T^{CMD} + \begin{pmatrix} 0 \\ \frac{1}{J^L} \\ 0 \\ \frac{K_{wd}}{J^L} \\ 0 \end{pmatrix} F_d(\omega^L) - \begin{pmatrix} d_1 \\ d_2 \\ d_3 \\ d_4 \\ d_5 \end{pmatrix} \omega^M$$

where d<sub>1</sub>-d<sub>5</sub>: Certain coefficients

J<sup>M</sup>: Inertia moment at Motor side

ω<sup>M</sup>: Angular velocity of Motor

R<sup>M</sup>: Pulley radius at Motor side

F: Tension of Belt

K<sub>b</sub>: Spring constant of Belt

J<sup>L</sup>: Inertia moment at Screw side

ω<sup>L</sup>: Angular velocity at Screw side

R<sup>L</sup>: Pulley radius at Screw side

F<sub>d</sub>(ω<sup>L</sup>): Dynamic frictional resistance at Screw side

K<sub>w</sub>: Elastic modulus of Resin

K<sub>wd</sub>: Coefficient of Viscosity of Resin

σ: Force of Screw pushing Resin

[5] (Original) The method of controlling pressure in an electric injection molding machine according to claim 1,

wherein said screw in said injection molding machine and said motor are coupled together via a belt suspended around pulleys mounted on respective rotation shafts, and

wherein said observer is represented by the following Expression 4.

[Expression 4]

$$\begin{aligned}
 \hat{\omega}^M &= \hat{\omega}^{M-1} + \left\{ d_1 (\hat{\omega}^{M-1} - \omega^M) + \frac{1}{J^M} (T^{CMD-1} - R^M \hat{F}_{-1}) \right\} dt \\
 \hat{\omega}^L &= \hat{\omega}^{L-1} + \left\{ d_2 (\hat{\omega}^{M-1} - \omega^M) + \frac{1}{J^L} (R^L \hat{F}_{-1} + \hat{\delta}_{-1} + F_d(\omega^L)) \right\} dt \\
 \hat{F} &= \hat{F}_{-1} + \left\{ d_3 (\hat{\omega}^{M-1} - \omega^M) + K_b (R^M \hat{\omega}^{M-1} - R^L \hat{\omega}^{L-1}) \right\} dt \\
 \hat{\delta} &= \hat{\delta}_{-1} + \left\{ d_4 (\hat{\omega}^{M-1} - \omega^M) + K_w \hat{\omega}^{L-1} + \frac{K_{wd}}{J^L} (R^L \hat{F}_{-1} + \hat{\delta}_{-1} + F_d(\omega^L)) + \sigma_{-1} \right\} dt \\
 \hat{\sigma} &= \sigma_{-1} + d_5 (\hat{\omega}^{M-1} - \omega^M) dt
 \end{aligned}$$

where d<sub>1</sub>-d<sub>5</sub>: Certain coefficients

J<sup>M</sup>: Inertia moment at Motor side

ω<sup>M</sup>: Angular velocity of Motor

R<sup>M</sup>: Pulley radius at Motor side

F: Tension of Belt

K<sub>b</sub>: Spring constant of Belt

J<sup>L</sup>: Inertia moment at Screw side

ω<sup>L</sup>: Angular velocity at Screw side

$R^L$ : Pulley radius at Screw side

$F_d(\omega^L)$ : Dynamic frictional resistance at Screw side

$K_w$ : Elastic modulus of Resin

$K_{wd}$ : Coefficient of Viscosity of Resin

$\sigma$ : Force of Screw pushing Resin

$x_{-1}$ : Value of  $x$  at Immediately preceding processing period

[6] (Original) The method of controlling pressure in an electric injection molding machine according to claim 1,

wherein said screw in said injection molding machine and said motor are coupled together via a belt suspended around pulleys mounted on respective rotation shafts, and

wherein said observer is represented by the following Expression 5.

[Expression 5]

$$\frac{d}{dt} \begin{pmatrix} \hat{\omega}^M \\ \hat{\omega}^L \\ \hat{P} \\ \hat{\delta} \end{pmatrix} = \begin{pmatrix} d_1 & 0 & -\frac{R^M}{J^M} & 0 \\ d_2 & 0 & \frac{R^L}{J^L} & \frac{1}{J^L} \\ d_3 + K_b R^M & -K_b R^L & 0 & 0 \\ d_4 & 0 & 0 & 0 \end{pmatrix} \begin{pmatrix} \hat{\omega}^M \\ \hat{\omega}^L \\ \hat{P} \\ \hat{\delta} \end{pmatrix} + \begin{pmatrix} \frac{1}{J^M} \\ 0 \\ 0 \\ 0 \end{pmatrix} T^{CMD} + \begin{pmatrix} 0 \\ \frac{1}{J^L} \\ 0 \\ 0 \end{pmatrix} F_d(\omega^L) + \begin{pmatrix} d_1 \\ d_2 \\ d_3 \\ d_4 \end{pmatrix} \omega^M$$

where  $d_1$ - $d_4$ : Certain coefficients

$J^M$ : Inertia moment at Motor side

$\omega^M$ : Angular velocity of Motor

$R^M$ : Pulley radius at Motor side

$F$ : Tension of Belt

$K_b$ : Spring constant of Belt

$J^L$ : Inertia moment at Screw side

$\omega^L$ : Angular velocity at Screw side

$R^L$ : Pulley radius at Screw side

$F_d(\omega^L)$ : Dynamic frictional resistance at Screw side

[7] (Original) The method of controlling pressure in an electric injection molding machine according to claim 1,

wherein said screw in said injection molding machine and said motor are coupled together via a belt suspended around pulleys mounted on respective rotation shafts, and

wherein said observer is represented by the following Expression 6.

[Expression 6]

$$\begin{aligned}\hat{\omega}^M &= \hat{\omega}^{M-1} + \left\{ d_1 (\hat{\omega}^{M-1} - \omega^M) + \frac{1}{J^M} (r^{CMD-1} - R^M \hat{r}_{-1}) \right\} dt \\ \hat{\omega}^L &= \hat{\omega}^{L-1} + \left\{ d_2 (\hat{\omega}^{M-1} - \omega^M) + \frac{1}{J^L} (R^L \hat{r}_{-1} + \hat{\delta}_{-1} + F_d(\omega^L)) \right\} dt \\ \hat{r} &= \hat{r}_{-1} + \left\{ d_3 (\hat{\omega}^{M-1} - \omega^M) + K_b (R^M \hat{\omega}^{M-1} - R^L \hat{\omega}^{L-1}) \right\} dt \\ \hat{\delta} &= \hat{\delta}_{-1} + d_4 (\hat{\omega}^{M-1} - \omega^M) dt\end{aligned}$$

where  $d_1$ - $d_4$ : Certain coefficients

$J^M$ : Inertia moment at Motor side

$\omega^M$ : Angular velocity of Motor

$R^M$ : Pulley radius at Motor side

$F$ : Tension of Belt

$K_b$ : Spring constant of Belt

$J^h$ : Inertia moment at Screw side

$\omega^h$ : Angular velocity at Screw side

$R^h$ : Pulley radius at Screw side

$F_d(\omega^h)$ : Dynamic frictional resistance at Screw side

$x_{-1}$ : Value of  $x$  at Immediately preceding processing period

[8] (Original) The method of controlling pressure in an electric injection molding machine according to claim 3, 5 or 7, further comprising:

calculating said torque command value  $T^{CMD}$  for said motor based the following Expression 7; and

feeding back said torque command value to said motor.

$$T^{CMD} = k_p (\delta^{REF} - \delta^A) + \alpha$$

[Expression 7]

where  $k_p$ : Certain constant

$\alpha$ : Certain function or constant

[9] (Currently Amended) An apparatus for controlling pressure in an electric injection molding machine, comprising:

an observer arithmetic unit operative to derive an estimated melt pressure value  $\delta^{\wedge}$  without deriving a differential of the detected angular velocity  $\omega$ , based on an observer, from an angular velocity  $\omega$  of a motor operative to propel forward a screw in an injection molding machine and a torque command value  $T^{CMD}$  given to said motor; and

a torque arithmetic unit operative to calculate said torque command value  $T^{CMD}$  for said motor using said estimated melt pressure value  $\delta^{\wedge}$  derived at said observer arithmetic unit and feed back said torque command value to said motor.

[10] (Original) The method of controlling pressure in an electric injection molding machine according to claim 1, further comprising deriving a dynamic frictional resistance  $F(\omega)$  from a relation between a velocity or position and a torque or current value associated with said motor at the time of injection with no resin loaded.

[11] (Original) The method of controlling pressure in an electric injection molding machine according to claim 1, further comprising:



defining a dynamic frictional resistance  $F(\omega)$  as a sum of a velocity-dependent component and a load-dependent component;

deriving said velocity-dependent component of said dynamic frictional resistance from a relation between a velocity or position and a torque or current value associated with said motor at the time of air shot; and

deriving said load-dependent component of said dynamic frictional resistance from a relation between a torque or current value and a pressure value at the time of injection with a plugged nozzle.